

## Složeno naprezanje

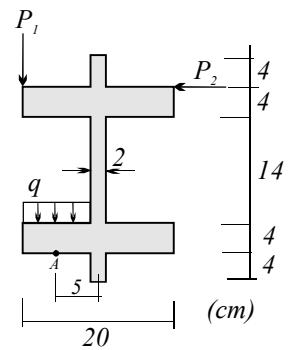
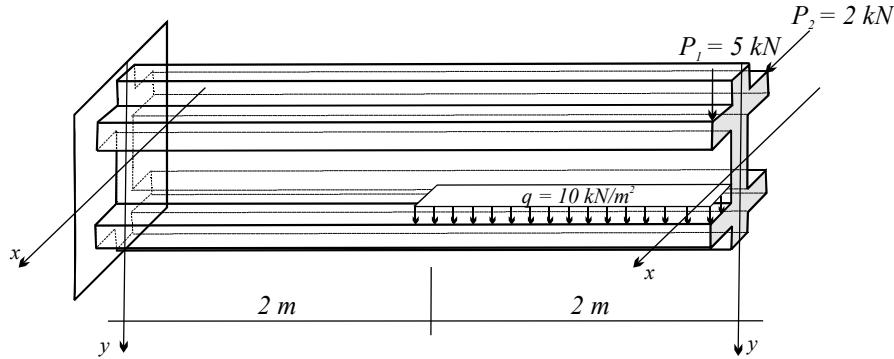
### Vežbe 6

Za nosač tankozidnog poprečnog preseka na slici:

a/ nacrtati dijagrame presečnih sila,

b/ za presek u uklještenju nacrtati dijagrame komponentalnih naponi,

c/ analizirati stanje napona u tački A.

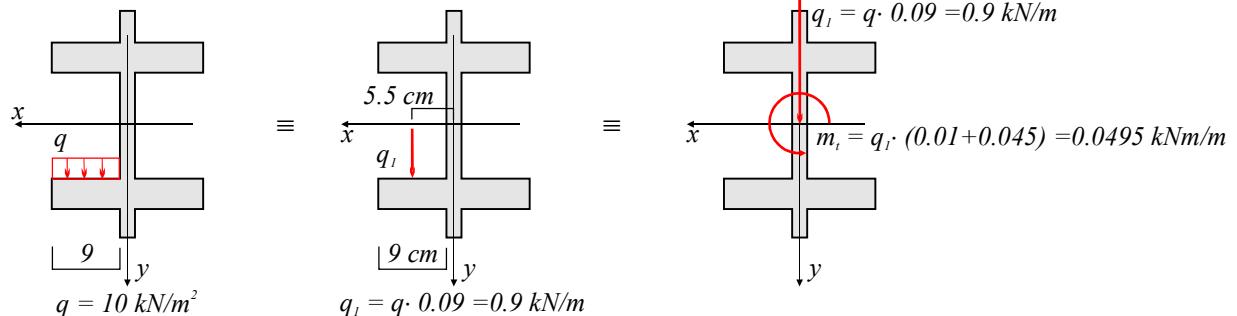


Rešenje:

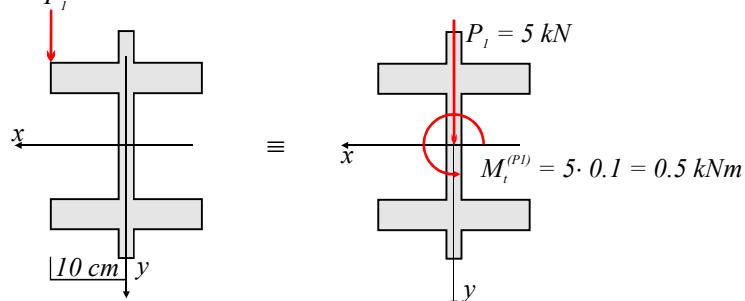
a/ Dijagrami presečnih sila:

Postupak redukcije opterećenja na težišnu osu.

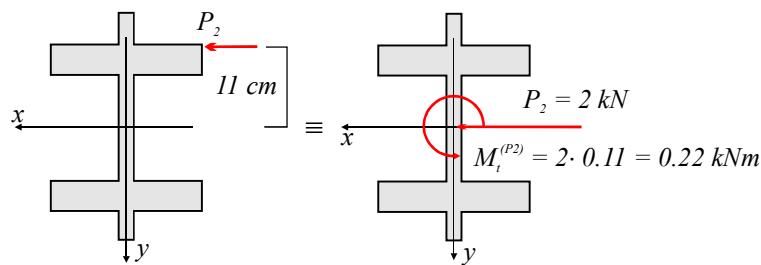
$$\text{Površinsko opterećenje } q = 10 \text{ kN/m}^2$$



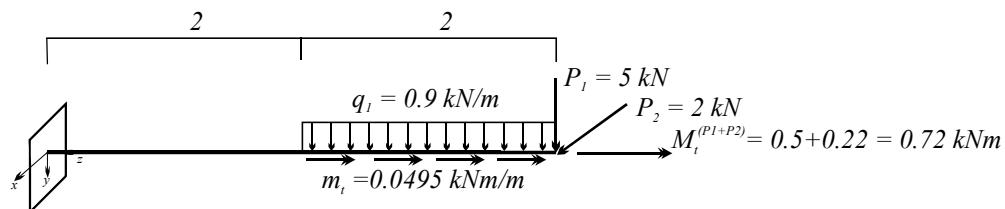
$$\text{Koncentrisana sila } P_1 = 5 \text{ kN}$$



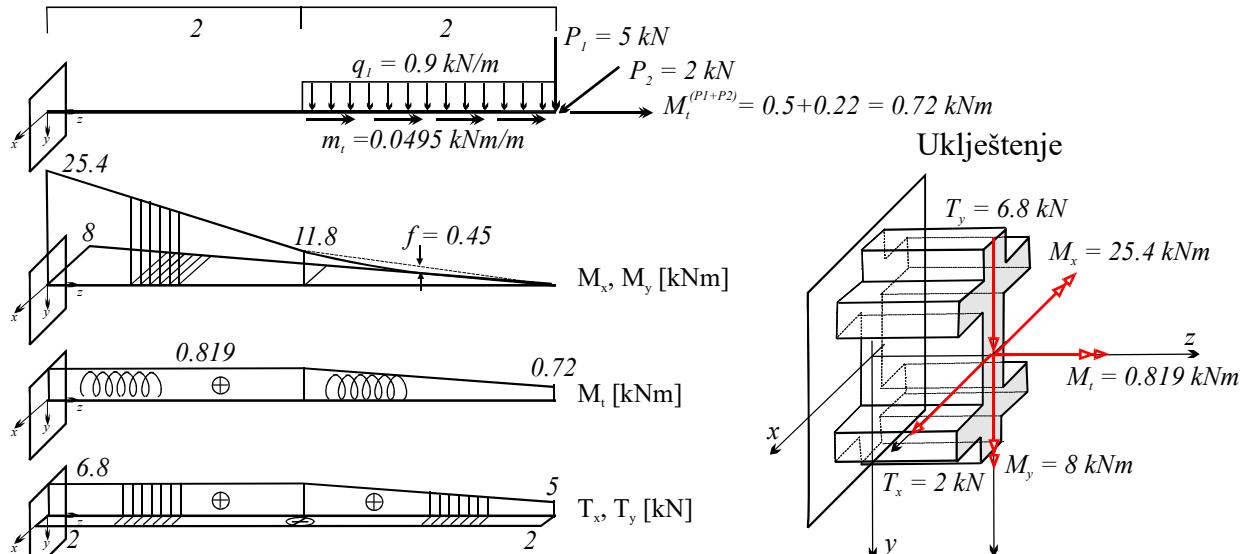
$$\text{Koncentrisana sila } P_2 = 2 \text{ kN}$$



Ekvivalentno opterećenje redukovano na težišnu osu nosača



## Dijagrami presečnih sila



Geometrijske karakteristike preseka

Momenti inercije ( $I_x, I_y$  i  $I_t$ )

$$I_x = \frac{1}{12} 2 \cdot 30^3 + 4 \left( \frac{1}{12} 9 \cdot 4^3 + 36 \cdot 9^2 \right) = 16356 \text{ cm}^4$$

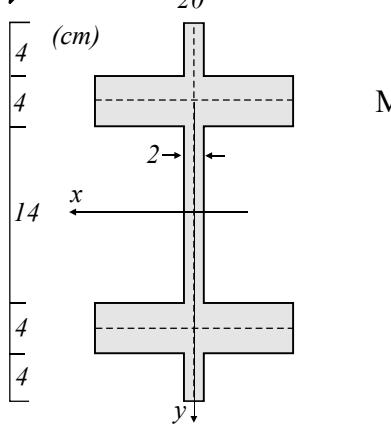
$$W_x = I_x / 15 = 1090.4 \text{ cm}^3$$

$$I_y = \frac{1}{12} 30 \cdot 2^3 + 4 \left( \frac{1}{12} 4 \cdot 9^3 + 36 \cdot 5.5^2 \right) = 5348 \text{ cm}^4$$

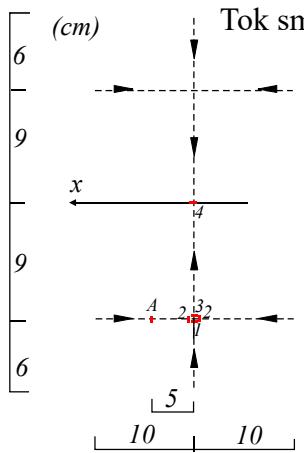
$$W_y = I_y / 10 = 534.8 \text{ cm}^3$$

$$I_t = \frac{1}{3} \sum_i h_i t_i^3 = \frac{1}{3} ( 30 \cdot 2^3 + 4 \cdot 10 \cdot 4^3 ) = 933.33 \text{ cm}^4$$

Statički momenti ( $S_x$  i  $S_y$ )



Tok smicanja



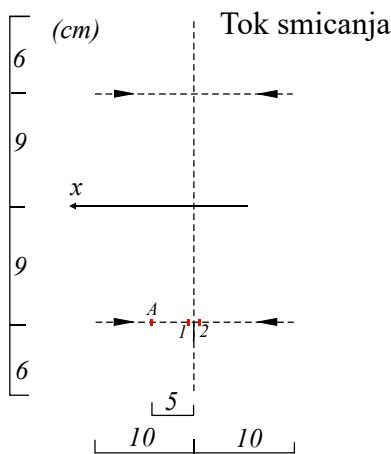
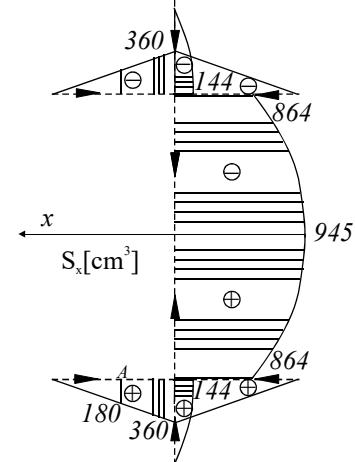
$$S_x^{(1)} = 6 \cdot 2 (9 + 3) = 144 \text{ cm}^3$$

$$S_x^{(4)} = 5 \cdot 4 \cdot 9 = 180 \text{ cm}^3$$

$$S_x^{(2)} = 10 \cdot 4 \cdot 9 = 360 \text{ cm}^3$$

$$S_x^{(3)} = S_x^{(1)} + 2 \cdot S_x^{(2)} = 864 \text{ cm}^3$$

$$S_x^{(4)} = S_x^{(3)} + 2 \cdot 9 \cdot \frac{9}{2} = 945 \text{ cm}^3$$

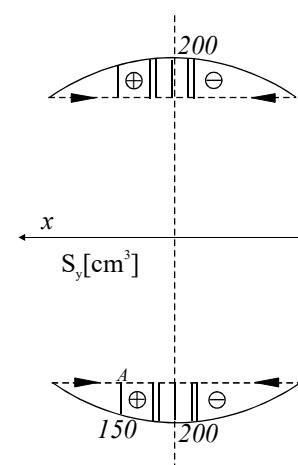


Tok smicanja

$$S_y^{(1)} = 10 \cdot 4 \frac{10}{2} = 200 \text{ cm}^3$$

$$S_y^{(4)} = 5 \cdot 4 \cdot 7.5 = 150 \text{ cm}^3$$

$$S_y^{(2)} = 10 \cdot 4 \cdot (-\frac{10}{2}) = -200 \text{ cm}^3$$



## Vrednosti komponentalnih naponova

Normalni naponi

Normalni naponi usled delovanja momenata savijanja  $M_x$  ( $M_x = 25.4 \text{ kNm}$ ) i  $M_y$  ( $M_y = 8 \text{ kNm}$ ):

$$\sigma_z^{(Mx)} = \pm \frac{M_x}{W_x} = \pm \frac{25.4 \cdot 10^2}{1090.4} = \pm 2.33 \text{ kN/cm}^2$$

$$\sigma_z^{(My)} = \pm \frac{M_y}{W_y} = \pm \frac{8 \cdot 10^2}{534.8} = \pm 1.496 \text{ kN/cm}^2$$

Komponente normalnog napona od momenata savijanja u tački A ( $y_A = 11 \text{ cm}$  i  $x_A = 5 \text{ cm}$ ). Pri određivanju vrednosti uvedena konvencija o znaku momenta savijanja.

$$\sigma_{z(A)}^{(Mx)} = \frac{M_x}{I_x} y_A = \frac{(-25.4) \cdot 10^2}{16356} 11 = -1.708 \text{ kN/cm}^2$$

$$\sigma_{z(A)}^{(My)} = \frac{M_y}{I_y} x_A = \frac{(-8) \cdot 10^2}{5348} 5 = -0.748 \text{ kN/cm}^2$$

Tangencijalni naponi

Smičući napon usled delovanja transverzalne sile  $T_y$  ( $T_y = + 6.8 \text{ kN}$ ):

$$\tau_{z(Ty)}^{(1)} = - \frac{T_y \cdot S_x^{(1)}}{b^{(1)} \cdot I_x} = - \frac{6.8 \cdot 144}{2 \cdot 16356} = -0.0299 \text{ kN/cm}^2$$

$$\tau_{z(Ty)}^{(2)} = - \frac{T_y \cdot S_x^{(2)}}{b^{(2)} \cdot I_x} = - \frac{6.8 \cdot 360}{4 \cdot 16356} = -0.0374 \text{ kN/cm}^2$$

$$\tau_{z(Ty)}^{(4)} = - \frac{T_y \cdot S_x^{(4)}}{b^{(4)} \cdot I_x} = - \frac{6.8 \cdot 180}{4 \cdot 16356} = -0.0187 \text{ kN/cm}^2$$

$$\tau_{z(Ty)}^{(3)} = - \frac{T_y \cdot S_x^{(3)}}{b^{(3)} \cdot I_x} = - \frac{6.8 \cdot 864}{2 \cdot 16356} = -0.1796 \text{ kN/cm}^2$$

$$\tau_{z(Ty)}^{(4)} = - \frac{T_y \cdot S_x^{(4)}}{b^{(4)} \cdot I_x} = - \frac{6.8 \cdot 945}{2 \cdot 16356} = -0.1964 \text{ kN/cm}^2$$

Smičući napon usled delovanja transverzalne sile  $T_x$  ( $T_x = + 2 \text{ kN}$ ):

$$\tau_{z(Tx)}^{(1)} = - \frac{T_x \cdot S_y^{(1)}}{b^{(1)} \cdot I_y} = - \frac{2 \cdot 200}{4 \cdot 5348} = -0.0187 \text{ kN/cm}^2$$

$$\tau_{z(Tx)}^{(2)} = - \frac{T_x \cdot S_y^{(2)}}{b^{(2)} \cdot I_y} = - \frac{2 \cdot (-200)}{4 \cdot 5348} = 0.0187 \text{ kN/cm}^2$$

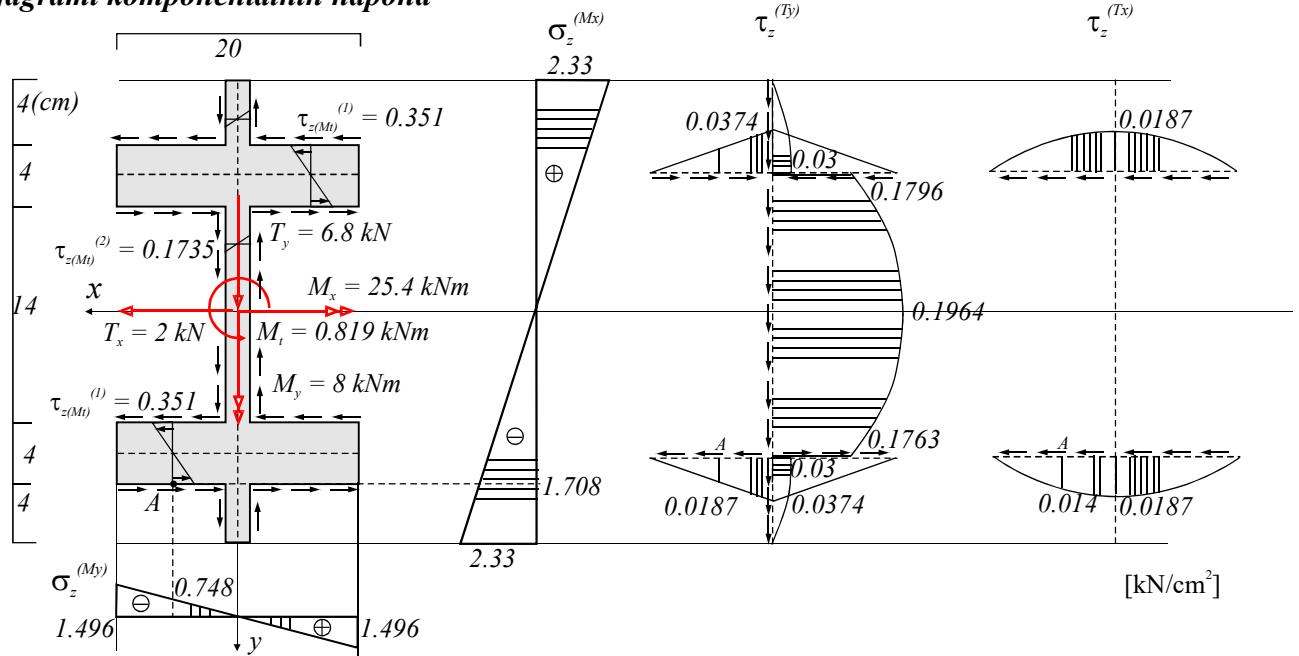
$$\tau_{z(Tx)}^{(4)} = - \frac{T_x \cdot S_y^{(4)}}{b^{(4)} \cdot I_y} = - \frac{2 \cdot 150}{4 \cdot 5348} = -0.0140 \text{ kN/cm}^2$$

Smičući napon usled delovanja momenta torzije  $M_t$  ( $M_t = 0.819 \text{ kNm}$ ):

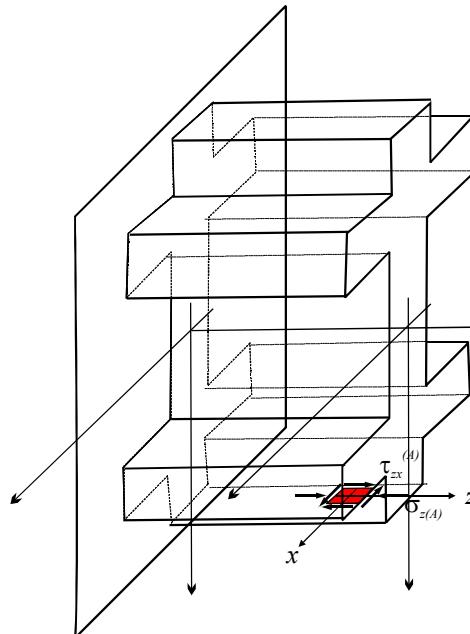
$$\tau_{z(Mt)}^{(1)} = \frac{M_t}{I_t} t_1 = \frac{0.819 \cdot 10^2}{933.33} 4 = 0.3510 \text{ kN/cm}^2$$

$$\tau_{z(Mt)}^{(2)} = \frac{M_t}{I_t} t_2 = \frac{0.819 \cdot 10^2}{933.33} 2 = 0.1755 \text{ kN/cm}^2$$

## Dijagrami komponentalnih napona



### Analiza napona u tački A



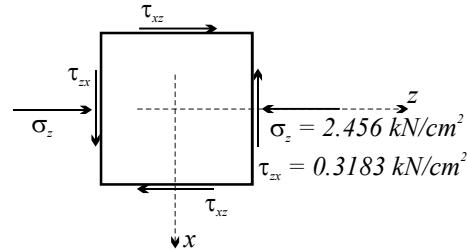
### Normalni napon u tački A

$$\sigma_{z(A)} = \sigma_{z(A)}^{(Mx)} + \sigma_{z(A)}^{(My)} = -1.708 - 0.748 = -2.456 \text{ kN/cm}^2$$

### Tangencijalni napon u tački A

$$\begin{aligned} \tau_{zx}^{(A)} &= \tau_{zx(T_x)}^{(A)} + \tau_{zx(T_y)}^{(A)} + \tau_{zx(M_t)}^{(A)} = \\ &= 0.014 + 0.0187 - 0.351 = -0.3183 \text{ kN/cm}^2 \end{aligned}$$

### Ulagani podaci pri analizi napona u tački A



### Glavni naponi i glavni pravci:

$$\sigma_{1/2} = 0.5 (\sigma_x + \sigma_z) \pm 0.5 \sqrt{(\sigma_x - \sigma_z)^2 + 4\tau_{xz}^2}$$

$$\sigma_{1/2} = 0.5(0 - 2.456) \pm 0.5 \sqrt{(0 + 2.453)^2 + 4 \cdot (-0.3183)^2} = -1.228 \pm 1.267$$

$$\begin{aligned} \sigma_1 &= 0.039 \text{ kN/cm}^2 \\ \sigma_2 &= -2.495 \text{ kN/cm}^2 \end{aligned}$$

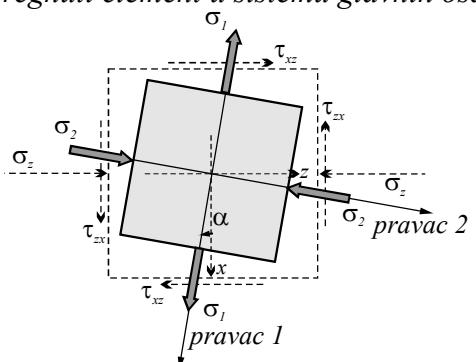
Pravci glavnih napona definisani su uglom  $\alpha$ :

$$\operatorname{tg} 2\alpha = \frac{2\tau_{xz}}{\sigma_x - \sigma_z} = \frac{2 \cdot (-0.3183)}{0 - (-2.453)} = -0.2595$$

$$za (\sigma_x - \sigma_z) > 0 \quad \alpha = 0.5 \operatorname{arctg} \left( \frac{2\tau_{xz}}{\sigma_x - \sigma_z} \right)$$

$$\alpha = 0.5 \operatorname{arctg} (-0.2595) = -7.274^\circ$$

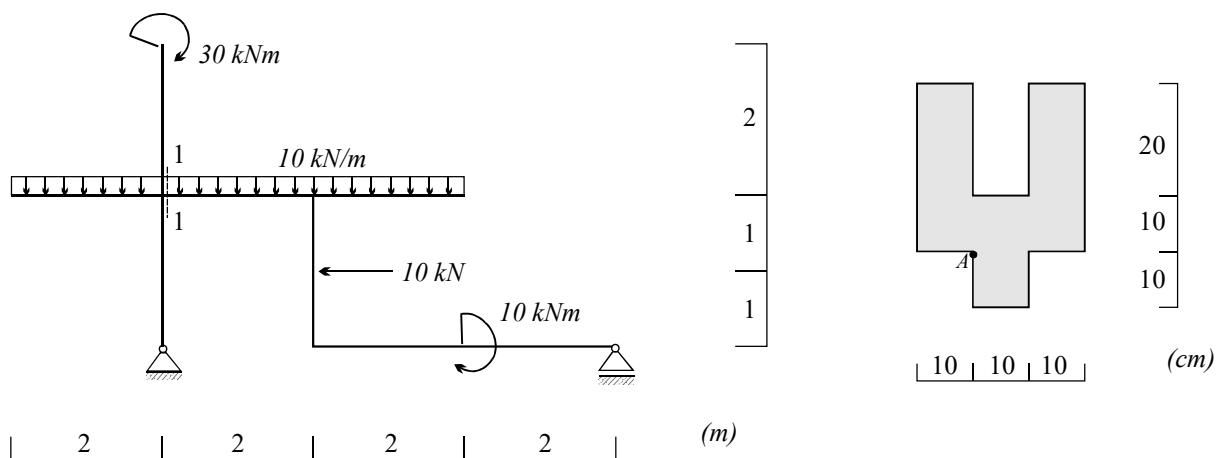
Napregnuti element u sistemu glavnih osa



Za nosač punog poprečnog preseka kao na slici:

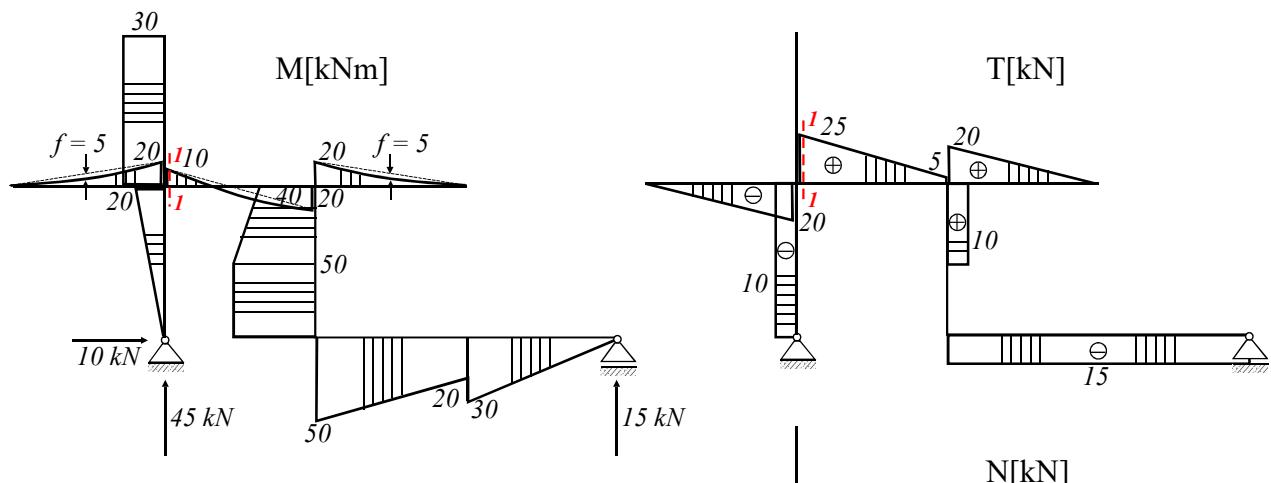
a/ nacrtati dijagrame presečnih sila ( $M$ ,  $N$ ,  $T$ ),

c/ za presek u 1-1 nacrtati dijagrame komponentalnih napona i analizirati stanje napona u tački A.

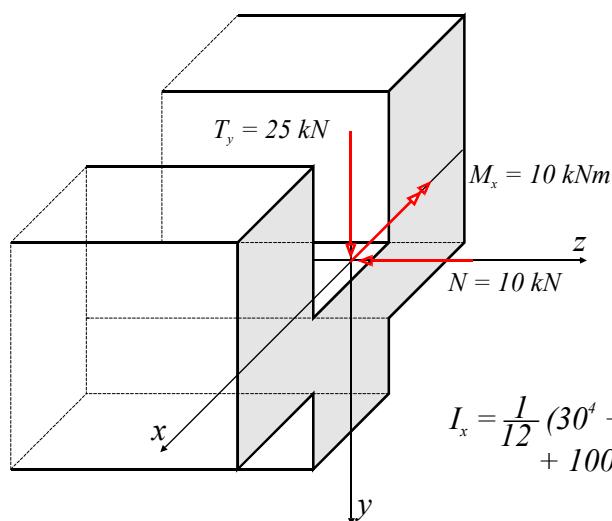


Rešenje:

a/ Dijagrami presečnih sila (smatra se da određivanje reakcija nosača u ravni nije potrebno detaljno prikazivati).



Uticaji u preseku 1-1



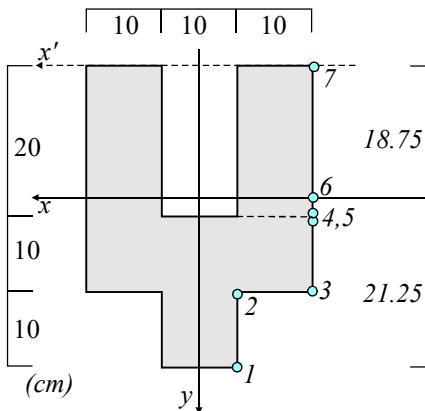
Geometrijske karakteristike preseka

Težište

$$y_c = \frac{900 \cdot 15 + 100 \cdot 35 - 200 \cdot 10}{800} = 18.75 \text{ cm}$$

Moment inercije u odnosu na x osu

$$I_x = \frac{1}{12} (30^4 + 10^4 - 10 \cdot 20^3) + 900 \cdot 3.75^2 + 100 \cdot 16.25^2 - 200 \cdot 8.75^2 = 85416.67 \text{ cm}^4$$



Statički moment

$$S_x^{(1)} = 0$$

$$S_x^{(2)} = S_x^{(3)} = 10 \cdot 10 \cdot 16.25 = 1625 \text{ cm}^3$$

$$S_x^{(4)} = S_x^{(5)} = S_x^{(3)} + 10 \cdot 30 \cdot 6.25 = 3500 \text{ cm}^3$$

$$S_x^{(6)} = S_x^{(5)} + 2 \cdot 10 \cdot 1.25 \cdot \frac{1.25}{2} = 3515.625 \text{ cm}^3$$

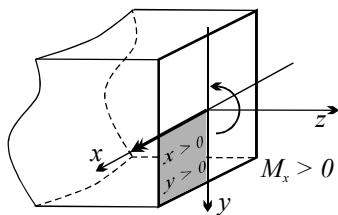
Kontrola

$$S_x^{(7)} = S_x^{(6)} - 2 \cdot 10 \cdot 18.75 \cdot \frac{18.75}{2} = 0$$

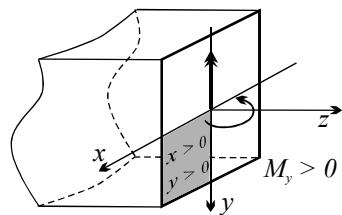
Normalni naponi

Normalni naponi usled delovanja momenata savijanja  $M_x$ .

Konvencija o znaku momenta savijanja pri određivanju intenziteta normalnog napona.



**Moment je pozitivan ako zateže kvadrant koji je usvojen kao prvi  $x > 0, y > 0$ .**



U ovom konkretnom slučaju moment  $M_x$  vrši pritisak na usvojeni prvi kvadrant zbog čega se u proračun uvodi sa negativnim predznakom ( $M_x = -10 \text{ kNm}$ ).

$$\sigma_{z(I)}^{(Mx)} = \frac{M_x}{I_x} y_I = \frac{(-10.0) \cdot 10^2}{85416.67} 21.25 = -0.249 \text{ kN/cm}^2$$

$$\sigma_{z(A)}^{(Mx)} = \frac{M_x}{I_x} y_A = \frac{(-10.0) \cdot 10^2}{85416.67} 11.25 = -0.132 \text{ kN/cm}^2$$

$$\sigma_{z(7)}^{(Mx)} = \frac{M_x}{I_x} y_7 = \frac{(-10.0) \cdot 10^2}{85416.67} (-18.75) = 0.2195 \text{ kN/cm}^2$$

Normalni naponi usled delovanja aksijalne sile pritiska  $N$ .

$$\sigma_z^{(N)} = \frac{N}{A} = \frac{-10}{800} = -0.0125 \text{ kN/cm}^2$$

Tangencijalni napon usled delovanja transverzalne sile  $T_y$  ( $T_y = 25 \text{ kN}$ ):

$$\tau_{zy}^{(1)} = 0 \quad i \quad \tau_{zy}^{(7)} = 0$$

$$\tau_{zy}^{(2)} = \frac{T \cdot S_x^{(2)}}{b^{(2)} \cdot I_x} = \frac{25 \cdot 1625}{10 \cdot 85416.67} = 0.0476 \text{ kN/cm}^2 = \tau_{zy}^{(4)}$$

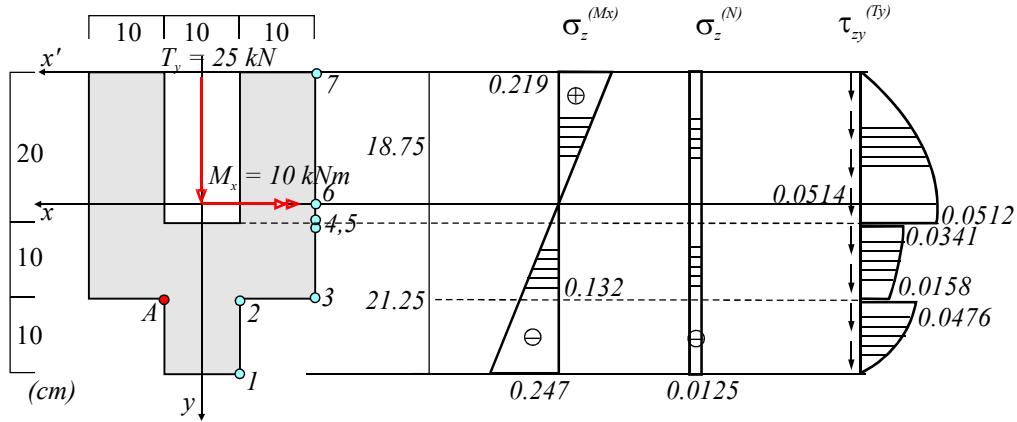
$$\tau_{zy}^{(3)} = \frac{T \cdot S_x^{(3)}}{b^{(3)} \cdot I_x} = \frac{25 \cdot 1625}{30 \cdot 85416.67} = 0.0158 \text{ kN/cm}^2$$

$$\tau_{zy}^{(4)} = \frac{T \cdot S_x^{(4)}}{b^{(4)} \cdot I_x} = \frac{25 \cdot 3500}{30 \cdot 85416.67} = 0.0341 \text{ kN/cm}^2$$

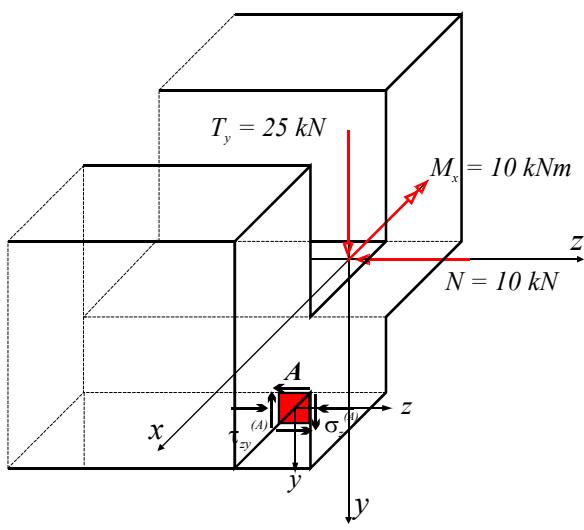
$$\tau_{zy}^{(5)} = \frac{T \cdot S_x^{(5)}}{b^{(5)} \cdot I_x} = \frac{25 \cdot 3500}{20 \cdot 85416.67} = 0.0512 \text{ kN/cm}^2$$

$$\tau_{zy}^{(6)} = \frac{T \cdot S_x^{(6)}}{b^{(6)} \cdot I_x} = \frac{25 \cdot 3515.625}{20 \cdot 85416.67} = 0.514 \text{ kN/cm}^2$$

## Dijagrami komponentalnih napona



Analiza napona u tački A



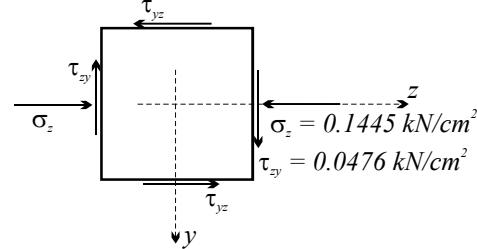
Normalni napon u tački A

$$\sigma_{z(A)} = \sigma_{z(A)}^{(Mx)} + \sigma_{z(A)}^{(N)} = -0.132 - 0.0125 \\ \sigma_{z(A)} = -0.1445 \text{ kN/cm}^2$$

Tangencijalni napon u tački A

$$\tau_{zy}^{(A)} = \tau_{zy(Ty)}^{(A)} = 0.0476 \text{ kN/cm}^2$$

Ulagani podaci pri analizi napona u tački A



Glavni naponi i glavni pravci:

$$\sigma_{1/2} = 0.5 (\sigma_y + \sigma_z) \pm 0.5 \sqrt{(\sigma_y - \sigma_z)^2 + 4\tau_{yz}^2}$$

$$\sigma_{1/2} = 0.5(0 - 0.1445) \pm 0.5 \sqrt{(0 + 0.1445)^2 + 4 \cdot 0.0476^2} = -0.07225 \pm 0.08652$$

$$\sigma_1 = 0.0143 \text{ kN/cm}^2$$

$$\sigma_2 = -0.1588 \text{ kN/cm}^2$$

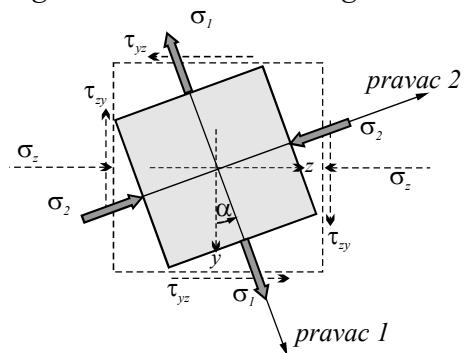
Pravci glavnih napona definisani su uglom  $\alpha$ :

$$\operatorname{tg} 2\alpha = \frac{2\tau_{yz}}{\sigma_y - \sigma_z} = \frac{2 \cdot 0.0476}{0 + 0.1445} = 0.659$$

$$\text{za } (\sigma_y - \sigma_z) > 0 \quad \alpha = 0.5 \operatorname{arctg} \left( \frac{2\tau_{yz}}{\sigma_y - \sigma_z} \right)$$

$$\alpha = 0.5 \operatorname{arctg} (0.659) = 16.69^\circ$$

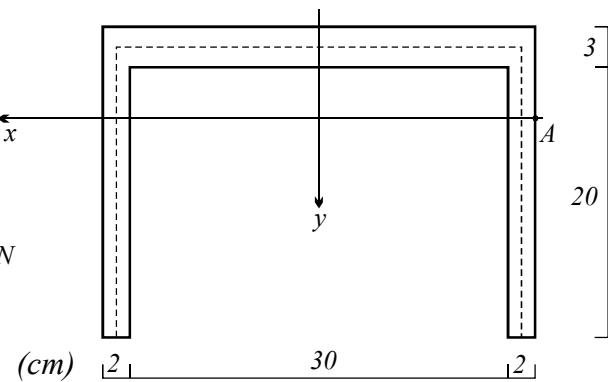
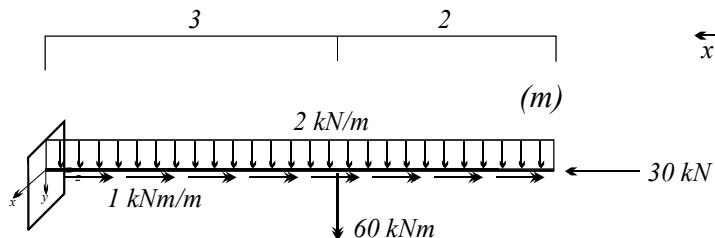
Napregnuti element u sistemu glavnih osa



Za nosač kao na slici:

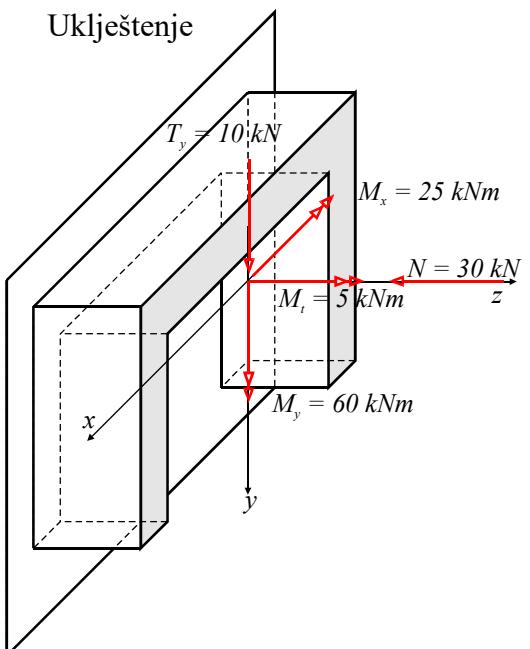
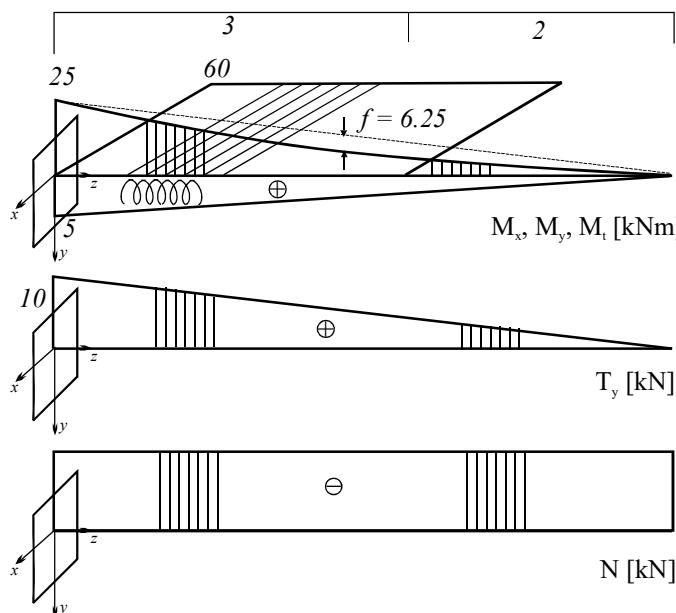
- nacrtati dijagrame presečnih sila ( $M$ ,  $N$ ,  $T$ ),
- za presek u ujleštenju nacrtati dijagrame komponentalnih napona,
- za tačku A

- napisati tenzor napona i deformacije ( $E=20 \text{ GPa}$ ,  $\nu=0.3$ ),
- izvršiti analizu napona,
- izračunati vrednost glavnih dilatacija.



Rešenje:

a/ Dijagrami presečnih sila

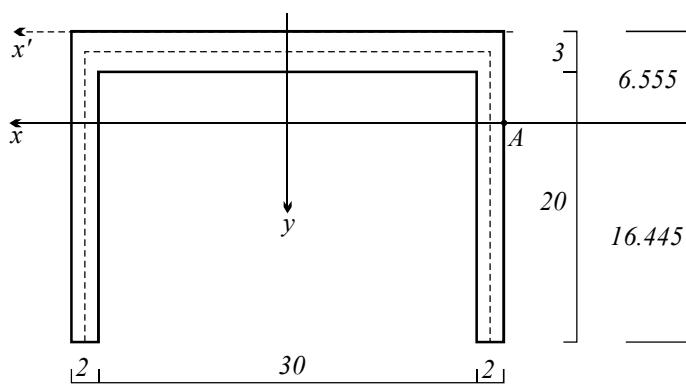


Geometrijske karakteristike preseka

$$A = 34 \cdot 23 - 30 \cdot 20 = 182 \text{ cm}^2$$

Težište

$$y_c = \frac{782 \cdot 11.5 - 600 \cdot 13}{182} = 6.555 \text{ cm}$$



Momenti inercije

$$I_x = \frac{1}{12} (34 \cdot 23^3 - 30 \cdot 20^3) + 782 \cdot 4.945^2 - 600 \cdot 6.445^2 = 8672.63 \text{ cm}^4$$

$$I_y = \frac{1}{12} (23 \cdot 34^3 - 20 \cdot 30^3) = 30332.67 \text{ cm}^4$$

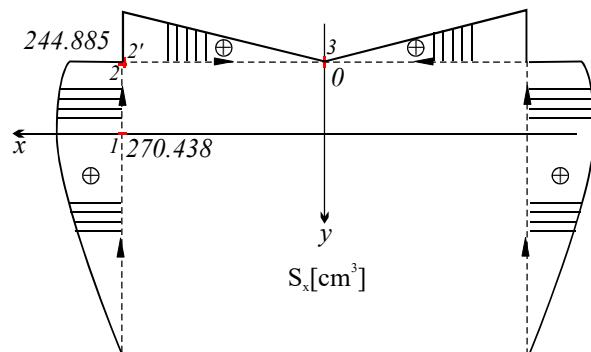
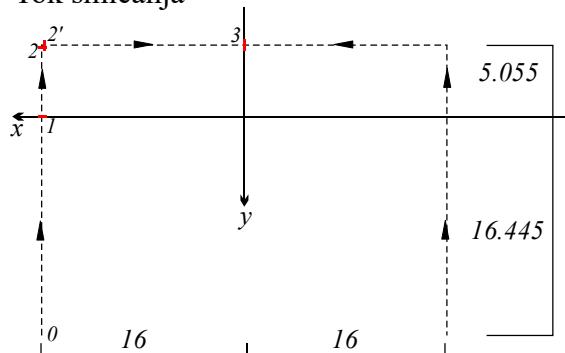
$$W_y = I_y / 17 = 1784.27 \text{ cm}_3$$

$$I_t = \frac{1}{3} \sum_i h_i t_i^3 = \frac{1}{3} (32 \cdot 3^3 + 2 \cdot 21.5 \cdot 2^3)$$

$$I_t = 402.67 \text{ cm}^4$$

## Statički momenti $S_x$

Tok smicanja



$$S_x^{(0)} = 0$$

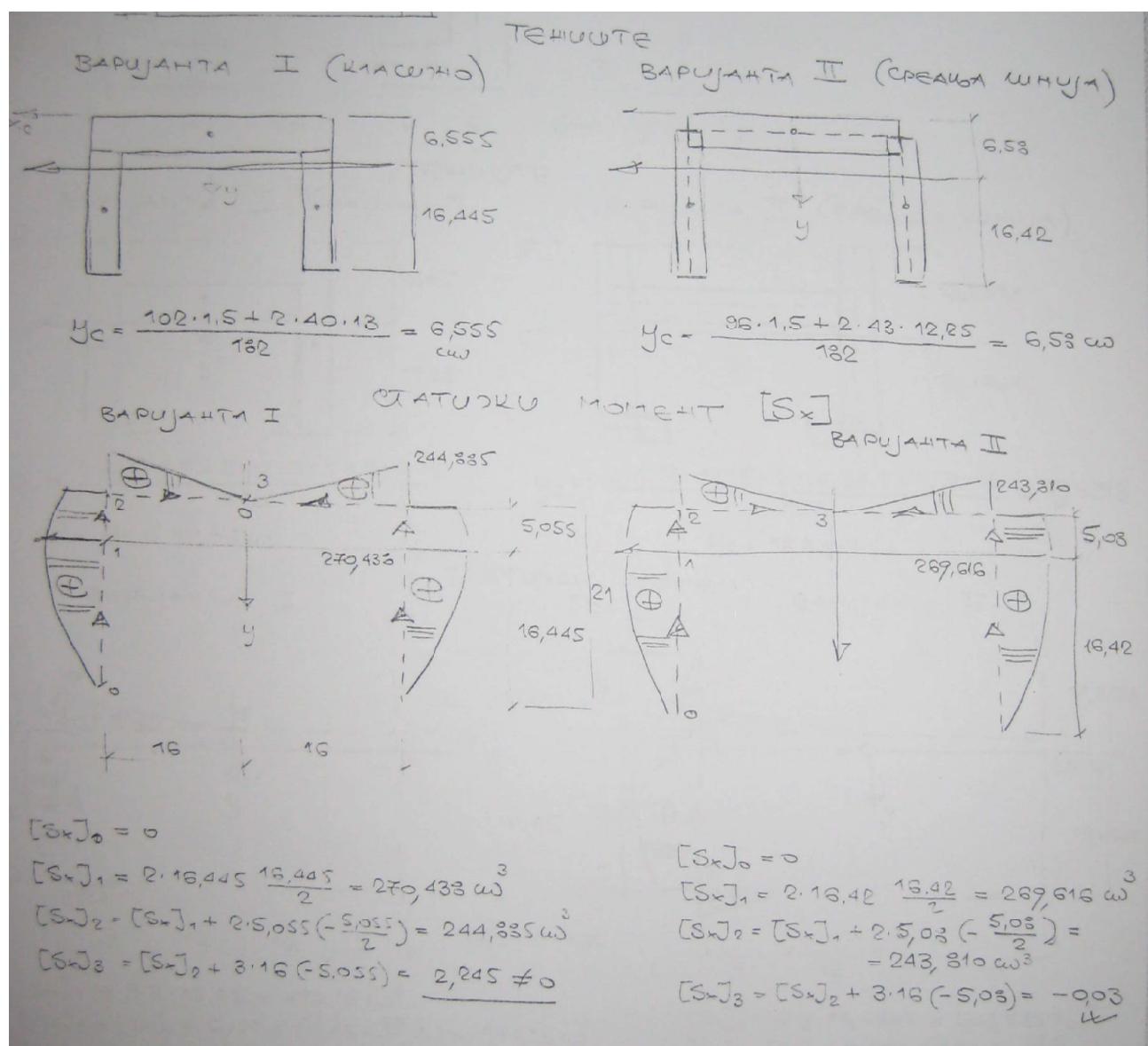
$$S_x^{(1)} = 16.445 \cdot 2 \cdot \frac{16.445}{2} = 270.438 \text{ cm}^3 = S_x^{(4)}$$

$$S_x^{(2)} = S_x^{(2)} = S_x^{(1)} - 5.055 \cdot 2 \cdot \frac{5.055}{2} = 244.885 \text{ cm}^3$$

$$\text{Kontrola } S_x^{(3)} = S_x^{(2)} - 3 \cdot 16 \cdot 5.055 = 244.885 - 242.640 = 2.245 \neq 0$$

$$S_x^{(3)} = 0$$

U tački 3 statički moment bi trebao biti jednak nuli, ali zbog razlike u postupku određivanja težišta u slučaju punih i tankozidnih preseka pojavljuje se odstupanje. S obzirom da se težište neće određivati posebno po principu tankozidnih preseka, nastala razlika se zanemaruje i u tački 3 se usvaja nulta vrednost statičkog momenta. U nastavku objašnjen je razlog nastanka ove razlike, koja u slučaju većeg razmimoilaženja u vrednostima težišta, može biti i izraženija.



## Vrednosti komponentalnih napona

Normalni naponi

Normalni naponi usled delovanja momenata savijanja  $M_x$  ( $M_x = 25 \text{ kNm}$ ) i  $M_y$  ( $M_y = 60 \text{ kNm}$ ):

$$\sigma_{z(0)}^{(Mx)} = \frac{M_x}{I_x} y_0 = \frac{(-25.0) \cdot 10^2}{8672.63} 16.445 = -4.740 \text{ kN/cm}^2$$

$$\sigma_{z(2)}^{(Mx)} = \frac{M_x}{I_x} y_2 = \frac{(-25.0) \cdot 10^2}{8672.63} (-6.555) = 1.890 \text{ kN/cm}^2$$

$$\sigma_z^{(My)} = \pm \frac{M_y}{W_y} = \pm \frac{60 \cdot 10^2}{1784.27} = \pm 3.363 \text{ kN/cm}^2$$

Normalni naponi usled delovanja aksijalne sile pritiska  $N$ .

$$\sigma_z^{(N)} = \frac{N}{A} = \frac{-30}{182} = -0.165 \text{ kN/cm}^2$$

Tangencijalni naponi

Smičući napon usled delovanja transverzalne sile  $T_y$  ( $T_y = +10 \text{ kN}$ ):

$$\tau_{z(Ty)}^{(1)} = -\frac{T_y \cdot S_x^{(1)}}{b^{(1)} \cdot I_x} = -\frac{10 \cdot 270.438}{2 \cdot 8672.63} = -0.156 \text{ kN/cm}^2 = \tau_{z(Ty)}^{(4)}$$

$$\tau_{z(Ty)}^{(2)} = -\frac{T_y \cdot S_x^{(2)}}{b^{(2)} \cdot I_x} = -\frac{10 \cdot 244.885}{2 \cdot 8672.63} = -0.141 \text{ kN/cm}^2$$

$$\tau_{z(Ty)}^{(2')} = -\frac{T_y \cdot S_x^{(2')}}{b^{(2')} \cdot I_x} = -\frac{10 \cdot 244.885}{3 \cdot 8672.63} = -0.094 \text{ kN/cm}^2$$

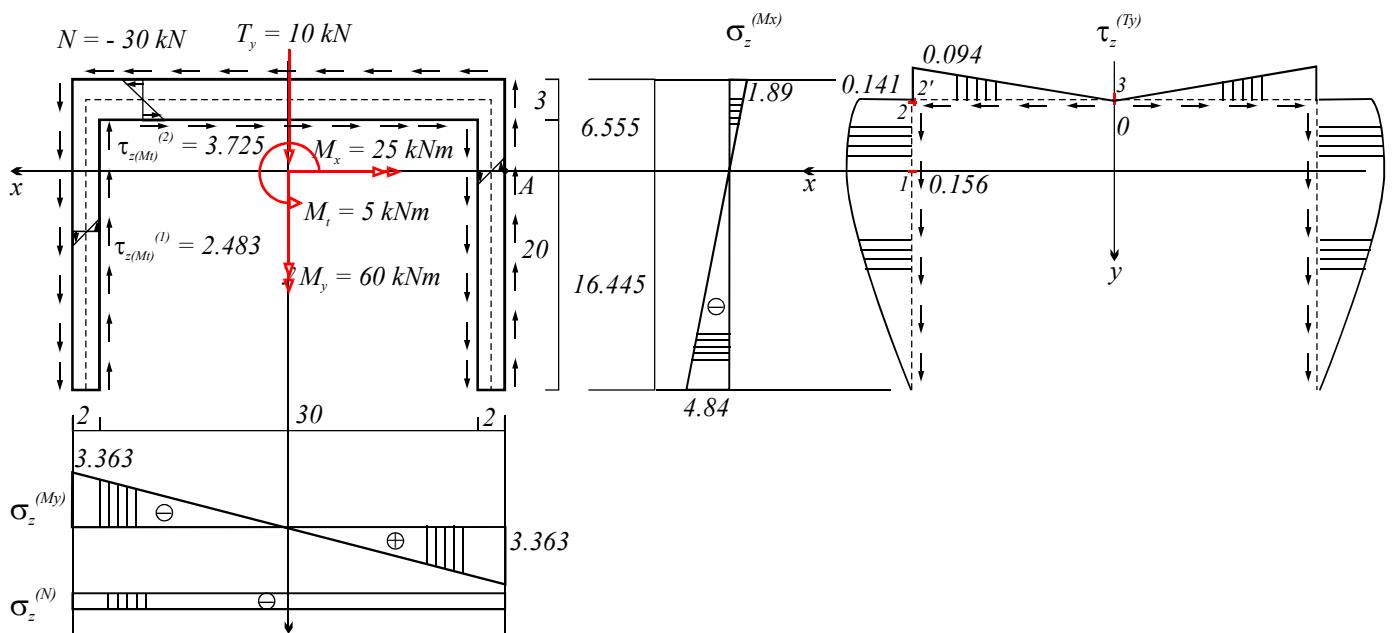
$$\tau_{z(Ty)}^{(3)} = 0$$

Smičući napon usled delovanja momenta torzije  $M_t$  ( $M_t = 0.819 \text{ kNm}$ ):

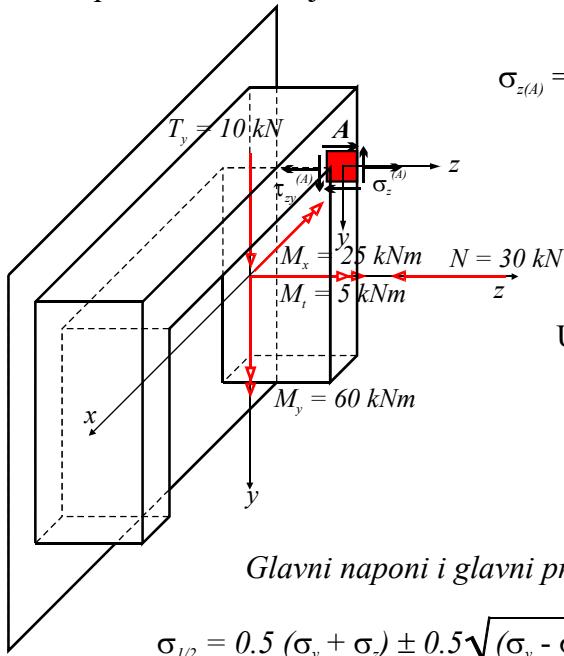
$$\tau_{z(Mt)}^{(1)} = \frac{M_t}{I_t} t_1 = \frac{5.0 \cdot 10^2}{402.67} 2 = 2.483 \text{ kN/cm}^2$$

$$\tau_{z(Mt)}^{(2)} = \frac{M_t}{I_t} t_2 = \frac{5.0 \cdot 10^2}{402.67} 3 = 3.725 \text{ kN/cm}^2$$

## Dijagrami komponentalnih napona



## Tenzori napona i deformacije u tački A



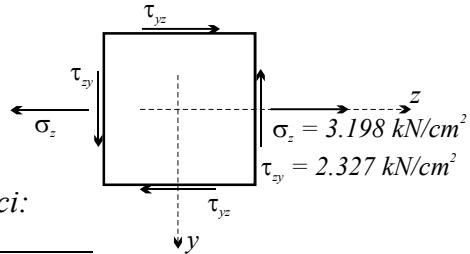
Normalni napon u tački A

$$\sigma_{z(A)} = \sigma_{z(A)}^{(Mx)} + \sigma_{z(A)}^{(My)} + \sigma_{z(A)}^{(N)} = 0 + 3.363 - 0.165 \\ \sigma_{z(A)} = 3.198 \text{ kN/cm}^2$$

Tangencijalni napon u tački A

$$\tau_{zy}^{(A)} = \tau_{zy(Ty)}^{(A)} + \tau_{zy(Mt)}^{(A)} = 0.156 - 2.483 \\ \tau_{zy}^{(A)} = -2.327 \text{ kN/cm}^2$$

Ulagani podaci pri analizi napona u tački A



Glavni naponi i glavni pravci:

$$\sigma_{1/2} = 0.5 (\sigma_y + \sigma_z) \pm 0.5 \sqrt{(\sigma_y - \sigma_z)^2 + 4\tau_{yz}^2}$$

$$\sigma_{1/2} = 0.5(0 + 3.198) \pm 0.5 \sqrt{(0 - 3.198)^2 + 4 \cdot (-2.327)^2} = 1.599 \pm 2.823$$

$$\sigma_1 = 4.422 \text{ kN/cm}^2 \\ \sigma_2 = -1.224 \text{ kN/cm}^2$$

Pravci glavnih napona definisani su uglom  $\alpha$ :

$$\operatorname{tg} 2\alpha = \frac{2\tau_{yz}}{\sigma_y - \sigma_z} = \frac{2 \cdot (-2.327)}{0 - 3.198} = 1.455$$

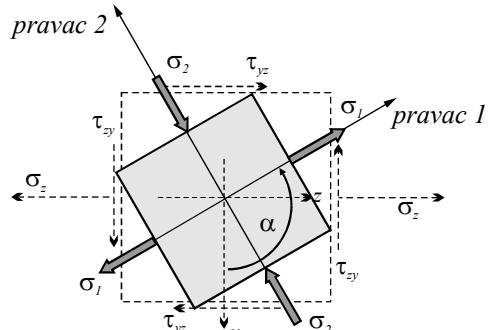
$$\text{za } (\sigma_y - \sigma_z) < 0 \quad \alpha = 0.5 \operatorname{arctg} \left( \frac{2\tau_{yz}}{\sigma_y - \sigma_z} \right) + 90^\circ$$

$$\alpha = 0.5 \operatorname{arctg} (1.455) + 90^\circ = 117.75^\circ$$

Tenzor napona

$$[S] = \begin{bmatrix} \sigma_x & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_y & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_z \end{bmatrix} = \begin{bmatrix} 0 & 0 & 0 \\ 0 & \sigma_y & \tau_{yz} \\ 0 & \tau_{yx} & \sigma_z \end{bmatrix} \quad [S] = \begin{bmatrix} \sigma_y & \tau_{yz} \\ \tau_{yx} & \sigma_z \end{bmatrix} = \begin{bmatrix} 0 & -2.327 \\ -2.327 & 3.198 \end{bmatrix} [\text{kN/cm}^2]$$

Napregnuti element u sistemu glavnih osa



Tenzor deformacija

$$E = 20 \text{ GPa} = 20 \cdot 10^9 \text{ kN/cm}^2 \quad G = \frac{E}{2(1+\nu)} = \frac{20}{2(1+0.3)} = 7.6923 \text{ GPa}$$

$$\varepsilon_x = [\sigma_x - \nu(\sigma_y + \sigma_z)]/E = \frac{1}{20 \cdot 10^9} [0 - 0.3 (0 + 3.198)] = -4.797 \cdot 10^{-4}$$

$$\varepsilon_y = [\sigma_y - \nu(\sigma_x + \sigma_z)]/E = \frac{1}{20 \cdot 10^9} [0 - 0.3 (0 + 3.198)] = -4.797 \cdot 10^{-4}$$

$$\varepsilon_z = [\sigma_z - \nu(\sigma_x + \sigma_y)]/E = \frac{1}{20 \cdot 10^9} [3.198 - 0.3 (0 + 0)] = 15.99 \cdot 10^{-4}$$

$$\gamma_{xy} = \tau_{xy}/G = 0$$

$$\gamma_{yz} = \tau_{yz}/G = \frac{-2.327}{7.6923 \cdot 10^9} = -30.251 \cdot 10^{-4}$$

$$\gamma_{zx} = \tau_{zx}/G = 0$$

tenzor deformacije

$$[D] = \begin{bmatrix} \varepsilon_x & \frac{1}{2}\gamma_{xy} & \frac{1}{2}\gamma_{xz} \\ \frac{1}{2}\gamma_{yx} & \varepsilon_y & \frac{1}{2}\gamma_{yz} \\ \frac{1}{2}\gamma_{zx} & \frac{1}{2}\gamma_{zy} & \varepsilon_z \end{bmatrix} = \begin{bmatrix} -4.797 & 0 & 0 \\ 0 & -4.797 & -\frac{1}{2}30.251 \\ 0 & -\frac{1}{2}30.251 & 15.99 \end{bmatrix} 10^{-4}$$

Glavne dilatacije je jednostavno odrediti preko glavnih napona ( $\sigma_1 > \sigma_2 > \sigma_3$ ).

$$\sigma_1 = 4.422 \text{ kN/cm}^2 \quad \sigma_2 = 0 \quad \sigma_3 = -1.224 \text{ kN/cm}^2$$

$$\varepsilon_1 = [\sigma_1 - v(\sigma_2 + \sigma_3)]/E = \frac{1}{20 \cdot 10^2} [4.422 - 0.3(0 - 1.224)] = 23.946 \cdot 10^{-4}$$

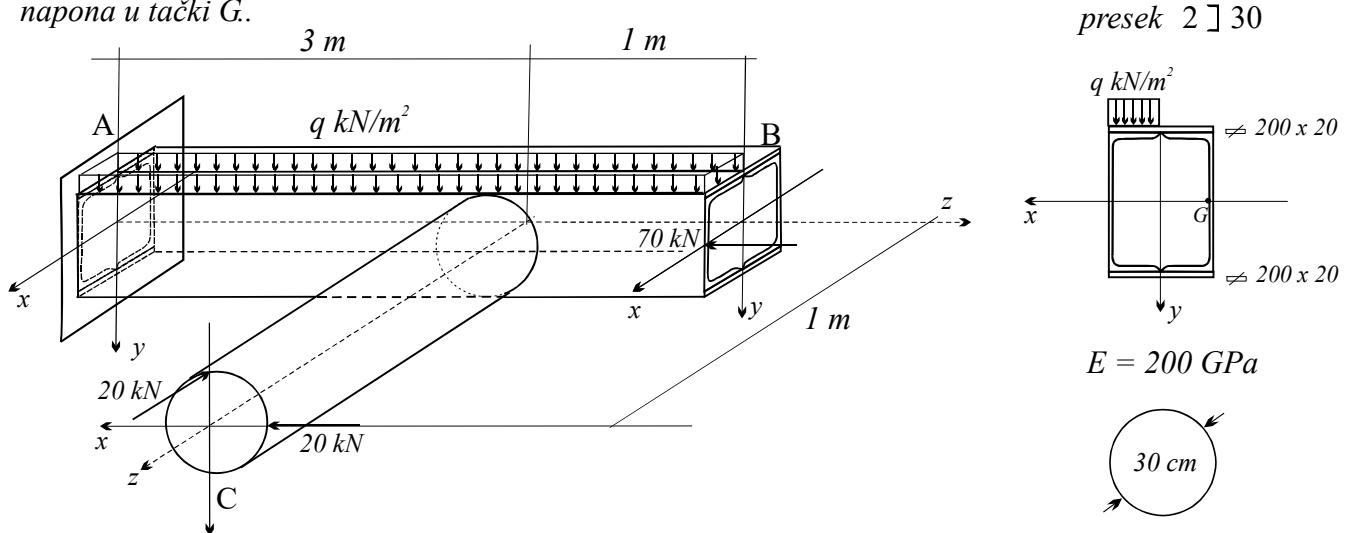
$$\varepsilon_2 = [\sigma_2 - v(\sigma_1 + \sigma_3)]/E = \frac{1}{20 \cdot 10^2} [0 - 0.3(4.422 - 1.224)] = -4.797 \cdot 10^{-4}$$

$$\varepsilon_3 = [\sigma_3 - v(\sigma_1 + \sigma_2)]/E = \frac{1}{20 \cdot 10^2} [-1.224 - 0.3(4.422 - 0)] = -12.753 \cdot 10^{-4}$$

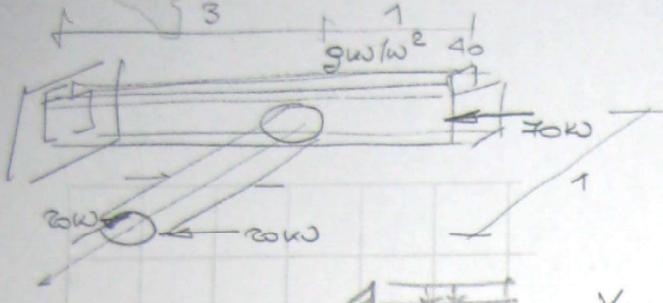
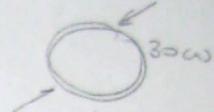
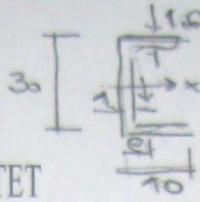
## ZADATAK ZA VEŽBU

Za konzolni nosač kao na slici:

- a/ odrediti vrednost opterećenja  $q$  tako da vertikalno pomeranje težišta preseka B bude 1.75 mm,
- b/ za tako dobijeno opterećenje nacrtati dijagrame presečnih sila ( $M, N, T$ ),
- c/ za presek u uklještenju nacrtati dijagrame komponentalnih napona i analizirati stanje napona u tački G.



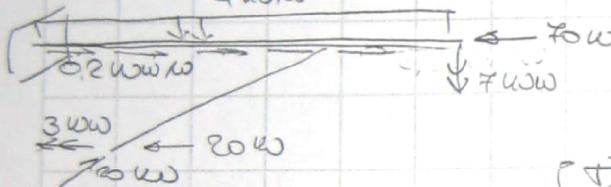
Priloženo je rešenje zadatka u skraćenom obliku



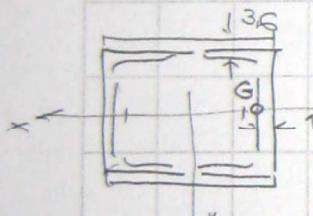
KONTROLA

$$\frac{0,1g \cdot 4^4 \cdot 16^2}{2 \cdot 800 \cdot 10 \cdot 36566,6} = 0,175$$

4 kN/w

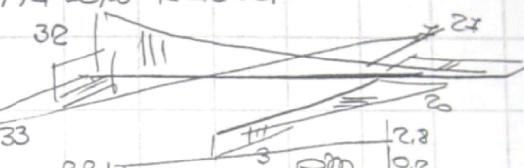


TEORIJSKI KAPACITET

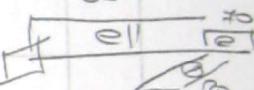


$U = -50 \text{ kN}$

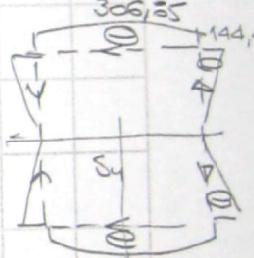
$$\begin{aligned} A &= 2 \cdot 50,2 + 2 \cdot 40 = 197,6 - 519,84 \\ I_x &= 36566,6 \text{ mm}^4 \\ I_y &= 2 \cdot 495 + 2 \cdot 50,2 \cdot 2,3 \\ &\quad + 2 \cdot \frac{1}{12} \cdot 2 \cdot 20^3 = 9923,57 \text{ mm}^4 \\ T_f &= \frac{4(197,6)}{2(3,6 + 1)} - 18701,936 \text{ mm}^4 \end{aligned}$$



[T]



N

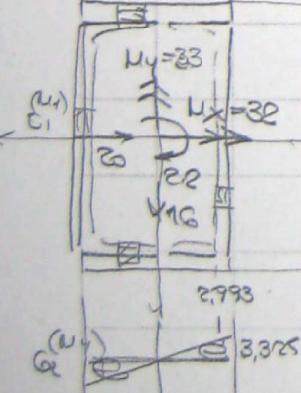


g

$\omega = -50 \text{ rad/s}$

$\omega_{\text{c}}$

$\omega_{\text{c}}^{(\text{u})}$



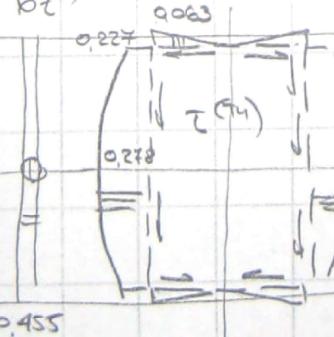
$\alpha_1^{(\text{u})}$

$\alpha_2^{(\text{u})}$

$\omega$

$\omega$

$\omega$



$\omega$

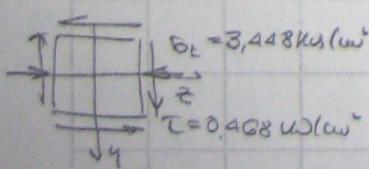
HANOTIĆ

$$\alpha_{\text{c}}^{(\text{u})} = \pm \frac{32 \cdot 16}{36566,6} \cdot 17 = \pm 1,49$$

$$\alpha_{\text{c}}^{(\text{u})} = \pm \frac{32 \cdot 10}{9923,57} \cdot 10 = \pm 3,38$$

$$\alpha_{\text{c}}^{(\text{u})} = - \frac{90}{197,6} = -0,455$$

TABLICA 6



$$G_c^6 = -2993 - 0,955 = -3,448$$

$$T = 0,18 + 0,278 = 0,468$$

$$\tau_1^{(\text{u})} = \frac{2,2 \cdot 10^2}{2(19,32,4)} = 0,19 \text{ kN/mm}$$

$$\tau_2^{(\text{u})} = \frac{2,2 \cdot 10}{2(19,32,4) \cdot 3,6} = 0,053 \text{ kN/mm}$$

$$\tau^{(\text{c})} = - \frac{(-20) \cdot 54}{6 \cdot 9923,57} = + 0,002015 \frac{54}{6} \text{ kN/mm}$$

$$\tau_1^{(\text{c})} = - \frac{(-144,4)}{1} = - 0,281 \text{ kN/mm}$$

$$\tau_1 = - \frac{(-144,4)}{3,6} = - 0,3889 \text{ kN/mm}$$

$$\tau_2 = - \frac{(-306,85)}{1} = - 0,172 \text{ kN/mm}$$

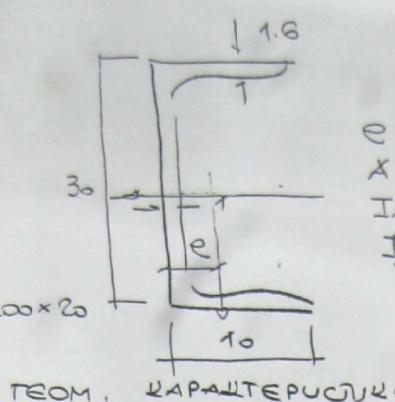
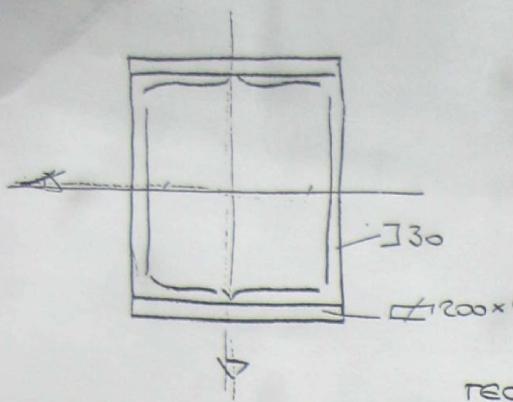
$$\tau^{(\text{c})} = - \frac{16 \cdot 54}{6 \cdot 36566,6} = - 0,000432 \text{ kN/mm}$$

$$\tau_1 = \sqrt{\frac{(-519,84)}{3,6}} = 0,063 \text{ kN/mm}$$

$$\tau_1 = \sqrt{\frac{(-519,84)}{1}} = 0,227 \text{ kN/mm}$$

$$\tau_2 = \sqrt{\frac{(-635,36)}{1}} = 0,278 \text{ kN/mm}$$

# GEOMETRIJSKE KARAKTERISTIKE SLOŽENIH ČELIČNIH PRESEKA

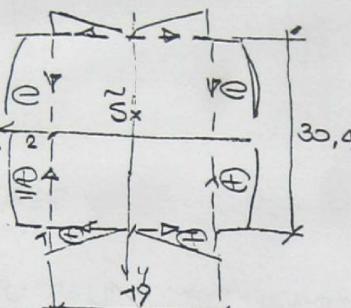
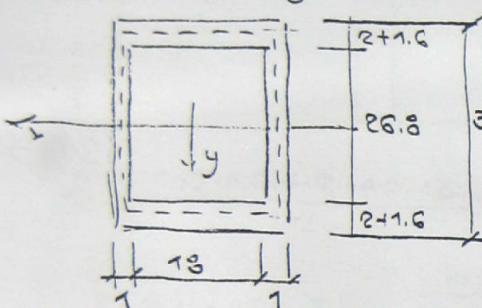


$$\begin{aligned} c &= 2.7 \omega \\ A &= 58.8 \omega^2 \\ I_x &= 8030 \omega^4 \\ I_y &= 495 \omega^4 \end{aligned}$$

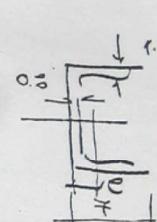
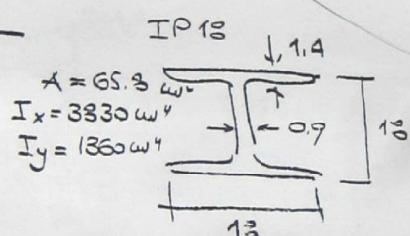
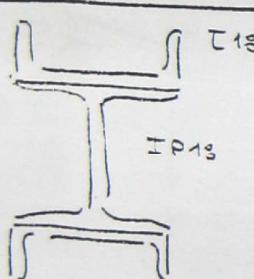
TEOM. KAPACITET PUCUJE

$$\begin{aligned} I_x &= 2 \cdot 8030 + 2 \left( \frac{1}{12} 20 \cdot 20^3 + 40 \cdot 16^2 \right) = 36566,6 \omega^4 \\ I_y &= 2(495 + 58.8 \cdot 1.6^2) + 2 \frac{1}{12} 2 \cdot 20^3 = 9523,57 \omega^4 \end{aligned}$$

CPEΔL6A MULHUJA

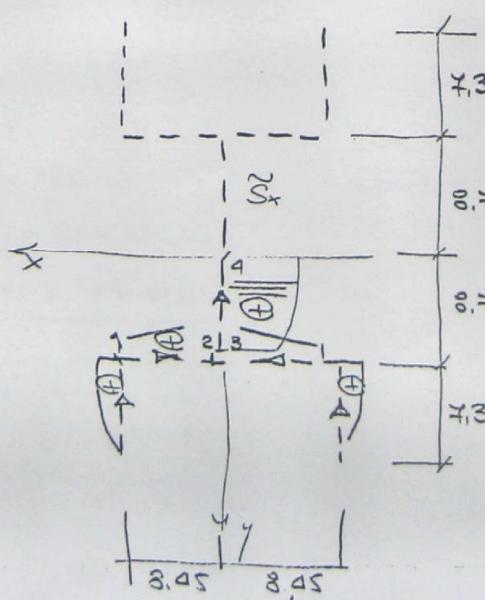
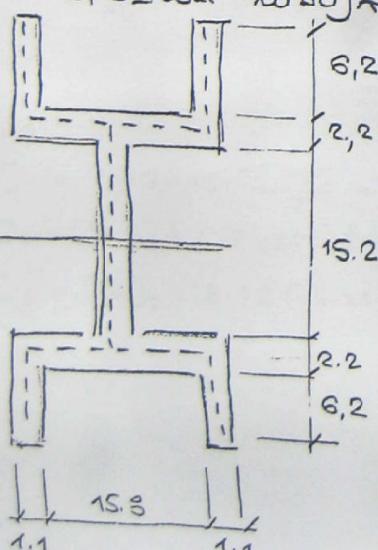


$$\begin{aligned} I_t &= \frac{2 A_s l}{2 \frac{h_i}{t_i}} = \frac{2 (19 \cdot 30,4)}{2 \left( \frac{19}{3,6} + \frac{30,4}{1} \right)} = \dots \\ [S_x]_1 &= 9,5 \cdot 3,6 \cdot 15,2 = 519,84 \omega^3 \\ [S_x]_2 &= 519,84 + 1 \cdot 15,2 \cdot \frac{15,2}{2} = 635,36 \omega^3 \end{aligned}$$



$$\begin{aligned} I_x &= 3830 + 2(114 + 28 \cdot 10,92^2) = 10735,8 \omega^4 \\ I_y &= 1360 + 2 \cdot 1350 = 4060 \omega^4 \end{aligned}$$

CPEΔL6A MULHUJA



$$[S_x]_1 = 2,3 \cdot 1,1 \cdot (3,7 + \frac{1,3}{2}) = 99,1705 \omega^3$$

$$[S_x]_2 = 99,1705 + 8,45 \cdot 2,2 \cdot 5,7 = 260,503 \omega^3$$

$$[S_x]_3 = 2[S_x]_2 = 521,807 \omega^3$$

$$[S_x]_4 = 521,807 + 0,9 \frac{8,7^2}{2} = 555,867 \omega^3$$

$$I_t = \frac{1}{3} \left[ 4 \cdot 7,3 \cdot 1,1^3 + 2 \cdot 16,9 \cdot 2,2^3 + 17,4 \cdot 0,9^3 \right] = 137,15 \omega^4$$